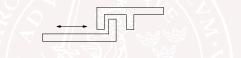
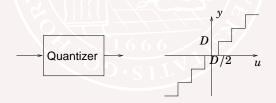
Lecture 8 — Backlash and Quantization

Today's Goal:

• To know models and compensation methods for backlash



• Be able to analyze the effect of quantization errors

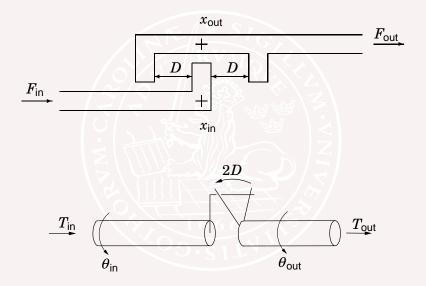


Material

Lecture slides

Note: We are using analysis methods from previous lectures (describing functions, small gain theorem etc.), and these have references to the course book(s).

Linear and Angular Backlash



Example: Parallel Kinematic Robot

Gantry-Tau robot: Need backlash-free gearboxes for high precision



EU-project: SMErobotTM www.smerobot.org

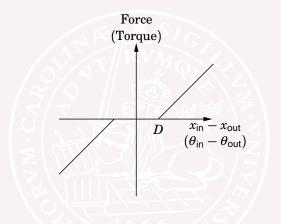
Backlash

Backlash (glapp) is

- present in most mechanical and hydraulic systems
- increasing with wear
- bad for control performance
- may cause oscillations

Note: A gear box without any backlash will not work if temperature rises

Dead-zone Model

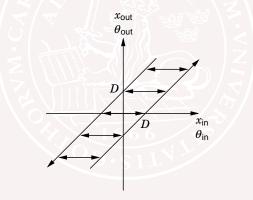


- Often easier to use model of the form $x_{in}(\cdot) \rightarrow x_{out}(\cdot)$
- Uses implicit assumption: F_{out} = F_{in}, T_{out} = T_{in}. Can be wrong, especially when "no contact".

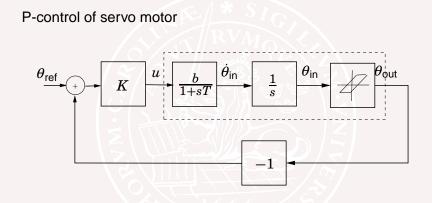
The Standard Model

Assume instead

- $\dot{x}_{out} = \dot{x}_{in}$ when "in contact"
- $\dot{x}_{out} = 0$ when "no contact"
- No model of forces or torques needed/used

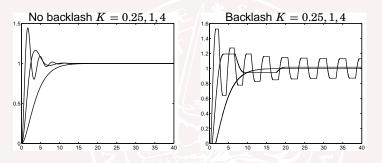


Servo motor with Backlash



How does the values of K and D affect the behavior?

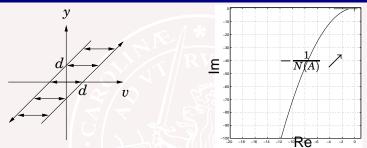
Effects of Backlash



Oscillations for K = 4 but not for K = 0.25 or K = 1. Why?

Limit cycle becomes smaller if D is made smaller, but it never disappears

Describing Function for a Backlash



If A > d then

$$N(A) = \frac{b_1 + ia_1}{A} \quad \text{with} \quad a_1 = \frac{4d}{\pi} \left(\frac{d}{A} - 1\right) \quad \text{and}$$
$$b_1 = \frac{A}{\pi} \left(\frac{\pi}{2} - \arcsin\left(\frac{2d}{A} - 1\right) + 2\left(1 - \frac{2d}{A}\right)\sqrt{\frac{d}{A}}\sqrt{1 - \frac{d}{A}}\right)$$

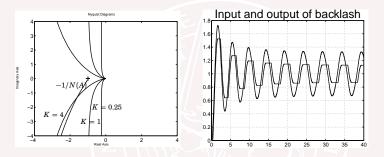
else N(A) = 0.

1 minute exercise

Study the plot for the describing function for the backlash on the previous slide.

Where does the function $-\frac{1}{N(A)}$ end for $A \to \infty$ and why?

Describing Function Analysis

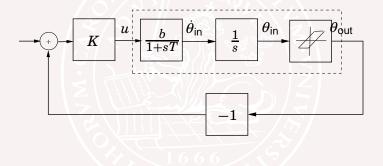


- For K = 4, D = 0.2: intersection between $G(j\omega)$ and -1/N(A) occurs for $A = 0.33, \omega = 1.24$
- Simulation: A = 0.33, $\omega = 2\pi/5.0 = 1.26$ Describing function predicts oscillation well!

Limit cycles?

The describing function method is only approximate.

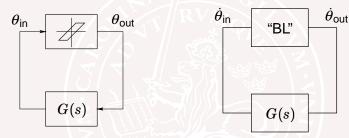
Can one determine conditions that guarantee stability?



Note: θ_{in} and θ_{out} will not converge to zero Idea: Consider instead $\dot{\theta}_{in}$ and $\dot{\theta}_{out}$

Backlash Limit Cycles

Rewrite the system as



Note that the block "BL" satisfies

$$\dot{\theta}_{out} = \begin{cases} \dot{\theta}_{in} & \text{in contact} \\ 0 & \text{otherwise} \end{cases}$$

Analysis by small gain theorem

Backlash block has gain ≤ 1 (from $\dot{\theta}_{in}$ to $\dot{\theta}_{out}$)

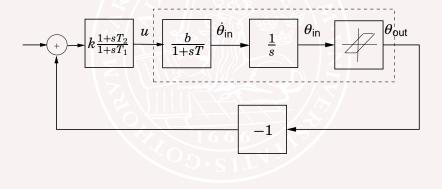
Hence closed loop is BIBO stable provided that G(s) is asymptotically stable and $|G(i\omega)| < 1$ for all ω

Backlash compensation

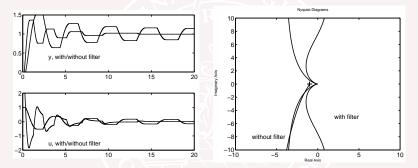
- Mechanical solutions
- Dead-zone
- Linear controller design
- Backlash inverse

Linear Controller Design

Introduce phase lead **to avoid** the -1/N(A) curve: Instead of only a P-controller we choose $K(s) = k \frac{1+sT_2}{1+sT_1}$

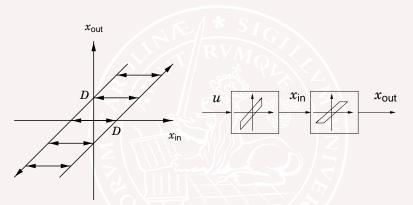


Controller $K(s) = k \frac{1+sT_2}{1+sT_1}$ Simulation with $T_1 = 0.5, T_2 = 2.0$



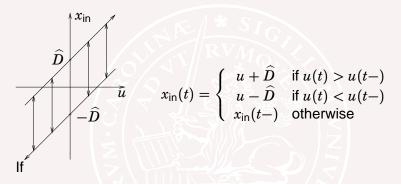
No limit cycle, oscillation removed!

Backlash Inverse



Idea: Let x_{in} jump $\pm 2D$ when \dot{x}_{out} should change sign. Works if the backlash is directly on the system input!

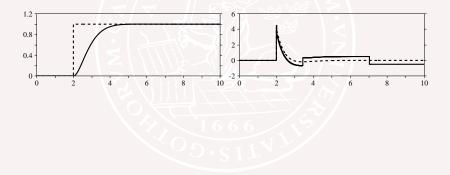
Backlash Inverse



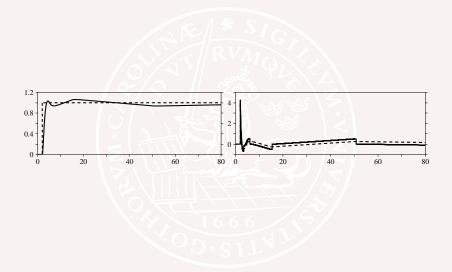
D
= D then x_{out}(t) = u(t) (perfect compensation)
 D
< D: Under-compensation (decreased backlash)
 D
> D: Over-compensation, often gives oscillations

Example–Perfect compensation

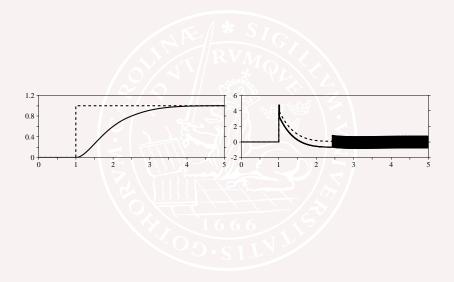
Motor with backlash on input, PD-controller



Example–Under compensation



Example–Over compensation



Backlash–More advanced models

Warning: More detailed models needed sometimes Model what happens when contact is attained Model external forces that influence the backlash Model mass/moment of inertia of the backlash.

Example: Parallel Kinematic Robot

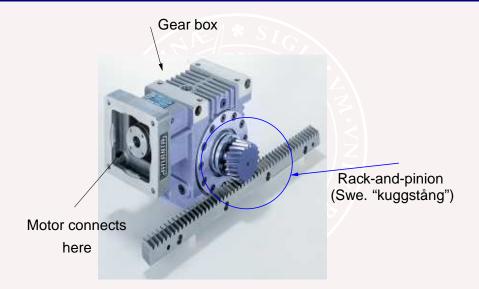
Gantry-Tau robot:

Need backlash-free gearboxes for very high precision

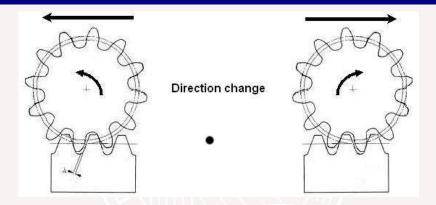


EU-project: SMErobotTM http://www.smerobot.org

"Rotational to Linear motion"



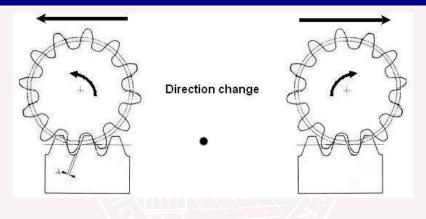
Backlash in gearbox and rails



Remedy:

Use two motors, possible to motors in opposite directions: One motor can act as spring and brake to "reduce" backlash. Need measurements on both motor and arm-side.

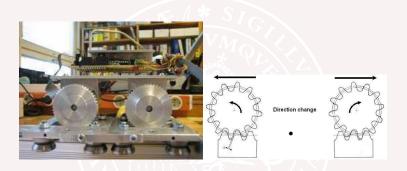
Backlash in gearbox and rails



Remedy:

Use two motors, possible to motors in opposite directions: One motor can act as spring and brake to "reduce" backlash. Need measurements on both motor and arm-side.

Backlash compensation



From master thesis by B. Brochier, *Control of a Gantry-Tau Structure, LTH, 2006* See also master theses by j. Schiffer and L. Halt, 2009.

Quantization



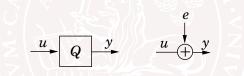
How accurate should the converters be? (8-14 bits?) What precision is needed in computations? (8-64 bits?)

- Quantization in A/D and D/A converters
- Quantization of parameters
- Roundoff, overflow, underflow in operations NOTE: Compare with (different) limits for "quantizer/dead-zone relay" in Lecture 6.

Linear Model of Quantization

Model the quantization error as a stochastic signal e independent of u with rectangular distribution over the quantization size.

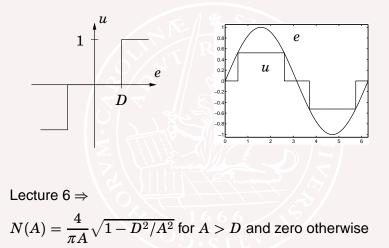
Works if quantization level is small compared to the variations in u



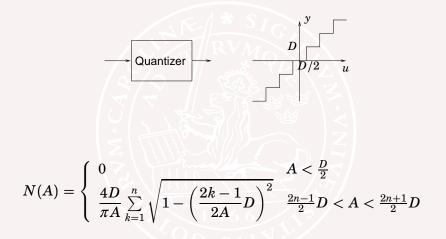
Rectangular noise distribution over $\left[-\frac{D}{2}, \frac{D}{2}\right]$ has the variance

$$Var(e) = \int_{-\infty}^{+\infty} e^2 f_e \, de = \int_{-D/2}^{D/2} e^2 \frac{1}{D} de = \frac{D^2}{12}$$

Describing Function for Deadzone Relay

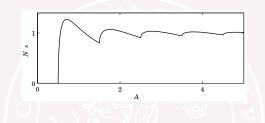


Describing Function for Quantizer



(See exercise)

Describing Function for Quantizer



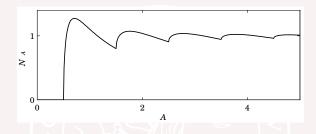
The maximum value is $4/\pi \approx 1.27$ for $A \approx 0.71D$.

Predicts limit cycle if Nyquist curve intersects negative real axis to the left of $-\pi/4 \approx -0.79$.

Should design for gain margin > 1/0.79 = 1.27!

Note that reducing D only reduces the size of the limit oscillation, the oscillation does not vanish completely.

5 minute exercise



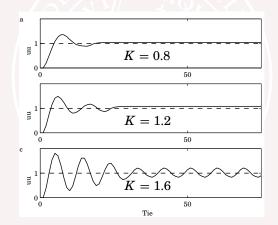
How does the shape of the describing function relate to the number of possible limit cycles and their stability.

What if the Nyquist plot

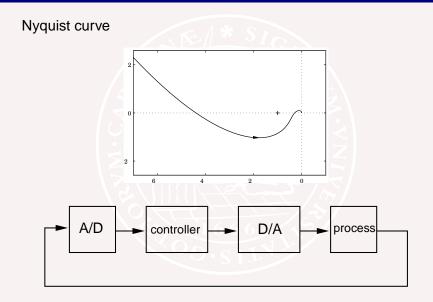
- intersects the negative real axis at -0.80?
- intersects the negative real axis at −1?
- intersects the negative real axis at -2?

Example – Motor with P-controller.

Roundoff at input, D = 0.2. Nyquist curve intersects at -0.5K. Hence stable for K < 2 without quantization. Stable oscillation predicted for K > 2/1.27 = 1.57.

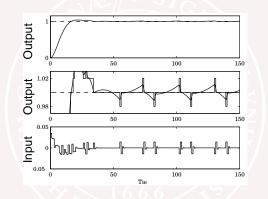


Example – Double integrator with 2nd order controller



Quantization at A/D converter

Double integrator with 2nd order controller, D = 0.02

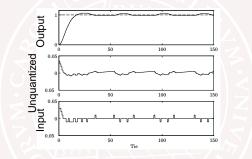


Describing function: $A_y \approx D/2 = 0.01$, period T = 39

Simulation: $A_y = 0.01$ and T = 28

Quantization at D/A converter

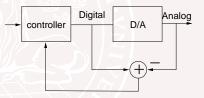
Double integrator with 2nd order controller, D = 0.01

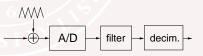


Describing function: $A_u \approx D/2 = 0.005$, period T = 39Simulation: $A_u = 0.005$ and T = 39Better prediction, since more sinusoidal signals

Quantization Compensation

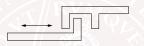
- Use improved converters, (small quantization errors/larger word length)
- Linear design, avoid unstable controller, ensure gain margin>1.3
- Use the tracking idea from anti-windup to improve D/A converter
- Use analog dither, oversampling and digital low-pass filter to improve accuracy of A/D converter





Today's Goal

To know models and compensation methods for backlash



Be able to analyze the effect of quantization errors

