Lecture 8

- Exact-linearization
- Lyapunov-based design
 - Lab 2
 - Adaptive control
 - Backstepping
- Hybrid / Piece-wise linear control
 - NOTE: Only overview!

Idea: Find state feedback u = u(x, v) so that nonlinear system

$$\dot{x} = f(x) + g(x)\iota$$

turns into linear system

$$\dot{x} = Ax + Bv$$

and then apply linear control design method.

Exact linearization: example [one-link robot]



$$m\ell^2\ddot{\theta} + d\dot{\theta} + m\ell g\cos\theta = u$$

where d is the viscous damping.

The control $u = \tau$ is the applied torque

Design state feedback controller u = u(x) with $x = (\theta, \dot{\theta})^T$

Introduce new control variable v and let

$$u = m\ell^2 v + d\dot{\theta} + m\ell g\cos\theta$$

 $\ddot{\theta} = v$

Then

Choose e.g. a PD-controller

$$v = v(\theta, \dot{\theta}) = k_p(\theta_{\mathsf{ref}} - \theta) - k_d \dot{\theta}$$

This gives the closed-loop system:

$$\ddot{\theta} + k_d \dot{\theta} + k_p \theta = k_p \theta_{\mathsf{ref}}$$

Hence, $u = m\ell^2 [k_p(\theta - \theta_{\text{ref}}) - k_d \dot{\theta}] + d\dot{\theta} + m\ell g \cos \theta$

Computed torque

The computed torque (also known as "Exact linearization", "dynamic inversion", etc.)

$$u = M(\theta)v + C(\theta, \dot{\theta})\dot{\theta} + G(\theta)$$

$$v = K_p(\theta_{ref} - \theta) - K_d\dot{\theta},$$
(1)

gives closed-loop system

$$\ddot{\theta} + K_d \dot{\theta} + K_p \theta = K_p \theta_{Ref}$$

The matrices K_d and K_p can be chosen diagonal (no cross-terms) and then this decouples into *n* independent second-order equations.

Identify FF and FB-part!

Block diagram



 $m\ell^2\ddot{\theta} + d\dot{\theta} + m\ell g\cos\theta = u$

$$\ddot{\theta} = \frac{1}{m\ell^2} \left(-d\dot{\theta} - m\ell g\cos\theta + u \right)$$

Multi-link robot (n-joints)



General form

$$M(\theta)\ddot{\theta} + C(\theta,\dot{\theta})\dot{\theta} + G(\theta) = u, \qquad \theta \in \mathbb{R}^n$$

Called *fully* actuated if *n* indep. actuators,

- M $n \times n$ inertia matrix, $M = M^T > 0$
- $C\dot{\theta}$ $n \times 1$ vector of centrifugal and Coriolis forces
- $n \times 1$ vector of centrificities to be $n \times 1$
- $G = n \times 1$ vector of gravitation terms

Cascade control - revisited

For systems with one control signal and many outputs:



- $G_{R_1}(s)$ controls the subsystem $G_{P_1}(s) \iff G_{y_1r_1}(s) \approx 1)$
- $G_{R_2}(s)$ controls the subsystem $G_{P_2}(s)$

Often used in motion control, e.g., robotics, with cascaded velocity and position controllers, BUT should have velocity reference feedforward!!

Example of couplings

"Robot Furuta pendulum": Underactuated - coupling as control action

velocity and torque reference (improved tracking!)

Lab 2 : Energy shaping for swing-up control

Often cascaded PI-controllers for each joint (inner velocity and outer position loop)

disturbance rejection between joints

Example: Couplings and interaction: "good"/"bad'

"Ordinary" Robot control:

Feedforward for



Lyapunov-Based Control Design Methods

$$\dot{x} = f(x, u)$$

- Find stabilizing state feedback u = u(x)
- Verify stability through Lyapunov function
- Methods depend on structure of f

Examples are energy shaping as in Lab 2 and e.g. **Back-stepping control design**, which require certain f discussed later.



Use Lyapunov-based design for swing-up control.

Lab 2 : Energy shaping for swing-up control

Consider the nonlinear system

$$\dot{x}_1 = -3x_1 + 2x_1x_2^2 + u \tag{2}$$
$$\dot{x}_2 = -x_2^3 - x_2,$$

Find a nonlinear feedback control law which makes the origin globally asymptotically stable.

Example of Lyapunov-based design

We try the standard Lyapunov function candidate

$$V(x_1, x_2) = rac{1}{2} \left(x_1^2 + x_2^2
ight),$$

which is radially unbounded, V(0,0) = 0, and $V(x_1, x_2) > 0 \ \forall (x_1, x_2) \neq (0, 0).$

Alt.2 Inserting the control law, $u = -2x_1x_2^2$, we get

$$\dot{V} = -3x_1^2 - x_2^2 \underbrace{-2x_1^2 x_2^2 + 2x_1^2 x_2^2}_{=0} - x_2^4 = -3x_1^2 - x_2^2 - x_2^4 < 0, \quad \forall x \neq 0$$

Both control alternatives gives global asymptotic stability of the origin.

Rough outline of method to get the pendulum to the upright position

- Find expression for total energy E of the pendulum (potential energy + kinetic energy)
- Let E_n be energy in upright position.
- Look at deviation $V = \frac{1}{2}(E E_n)^2 \ge 0$
- Find "swing strategy" of control torque u such that $\frac{dV}{dt} \leq 0$

Example - cont'd

$$\dot{V} = \dot{x_1}x_1 + \dot{x_2}x_2 = (-3x_1 + 2x_1x_2^2 + u)x_1 + (-x_2^3 - x_2)x_2$$

= $-3x_1^2 - x_2^2 + ux_1 + 2x_1^2x_2^2 - x_2^4$

We would like to have

$$\dot{V} < 0 \qquad \forall (x_1, x_2) \neq (0, 0)$$

Alt.1 Inserting the control law, $u = -x_1^3$, we get

$$\dot{V} = -3x_1^2 - x_2^2 - x_1^4 + 2x_1^2 x_2^2 - x_2^4 = -3x_1^2 - x_2^2 - \left(x_1^2 - x_2^2\right)^2 < 0, \quad \forall x \neq 0$$

Consider the system

$$\begin{aligned} \dot{x}_1 &= x_2^3 \\ \dot{x}_2 &= u \end{aligned} \tag{3}$$

Find a globally asymptotically stabilizing control law u = u(x). Attempt 1: Try the standard Lyapunov function **candidate**

$$V(x_1, x_2) = \frac{1}{2} \left(x_1^2 + x_2^2 \right)$$

which is radially unbounded, V(0,0) = 0, and $V(x_1, x_2) > 0 \ \forall (x_1, x_2) \neq (0, 0).$

$$\dot{V} = \dot{x_1}x_1 + \dot{x_2}x_2 = x_2^3 \cdot x_1 + u \cdot x_2 = x_2 \underbrace{(x_2^2x_1 + u)}_{-x_2} = -x_2^2 \le 0$$

where we chose

$$u = -x_2 - x_2^2 x_1$$

Attempt 2:

$$\begin{aligned} \dot{x}_1 &= x_2^3 \\ \dot{x}_2 &= u \end{aligned} \tag{5}$$

Try the Lyapunov function candidate

$$V(x_1, x_2) = \frac{1}{2}x_1^2 + \frac{1}{4}x_2^4,$$

which satisfies

- \blacktriangleright V(0,0) = 0
- ▶ $V(x_1, x_2) > 0$, $\forall (x_1, x_2) \neq (0, 0)$.

radially unbounded,

$$\frac{dV}{dt} = \dot{x}_1 x_1 + \dot{x}_2 x_2^3 = x_2^3 (x_1 + u) = -x_2^4 \le 0$$

 $u = -x_1 - x_2$ if we use $u = -x_1 - x_2$

Adaptive Noise Cancellation Revisited



$$\dot{x} + ax = bu$$
$$\dot{x} + \hat{a}\hat{x} = \hat{b}u$$

Introduce $\tilde{x} = x - \hat{x}$, $\tilde{a} = a - \hat{a}$, $\tilde{b} = b - \hat{b}$. Want to design adaptation law so that $\tilde{x} \to 0$

Back-Stepping Control Design

We want to design a state feedback u = u(x) that stabilizes

$$\dot{x}_1 = f(x_1) + g(x_1)x_2$$

 $\dot{x}_2 = u$ (7)

at x = 0 with f(0) = 0.

Idea: See the system as a cascade connection. Design controller first for the inner loop and then for the outer.



However $\dot{V} = 0$ as soon as $x_2 = 0$ (Note: x_1 could be anything).

According to LaSalle's theorem the set $E = \{x | \dot{V} = 0\} = \{(x_1, 0)\} \forall x_1$

What is the largest invariant set M?

Plugging in the control law $u = -x_2 - x_2^2 x_1$, we get

$$\begin{aligned} \dot{x}_1 &= x_2^3 \\ \dot{x}_2 &= -x_2 - x_2^2 x_1 \end{aligned} \tag{4}$$

(6)

and we see that if we start anywhere on the line $\{(x_1, 0)\}$ we will stay in the same point as both $\dot{x}_1 = 0$ and $\dot{x}_2 = 0$, thus M=E and we will not converge to the origin, but get stuck on the line $x_2 = 0$.

Draw phase-plot with e.g., pplane and study the behaviour.

With

$$u = -x_1 - x_2$$

we get the dynamics $\dot{x}_1 = x_2^3$

 $\dot{V} = 0$ if $x_2 = 0$, thus

$$E = \{x | \dot{V} = 0\} = \{(x_1, 0)\} \,\forall x_1$$

 $\dot{x}_2 = -x_1 - x_2$

However, now the only possibility to stay on $x_2 = 0$ is if $x_1 = 0$, (else $\dot{x}_2 \neq 0$ and we will leave the line $x_2 = 0$). Thus, the largest invariant set

$$M = (0, 0)$$

According to the Invariant Set Theorem (LaSalle) all solutions will end up in M and so the origin is GAS.

Draw phase-plot with e.g., pplane and study the behaviour.

Let us try the Lyapunov function

$$\begin{split} V &= \frac{1}{2} (\widetilde{x}^2 + \gamma_a \widetilde{a}^2 + \gamma_b \widetilde{b}^2) \\ \dot{V} &= \widetilde{x} \widetilde{x} + \gamma_a \widetilde{a} \widetilde{a} + \gamma_b \widetilde{b} \widetilde{b} = \\ &= \widetilde{x} (-a \widetilde{x} - \widetilde{a} \widetilde{x} + \widetilde{b} u) + \gamma_a \widetilde{a} \widetilde{a} + \gamma_b \widetilde{b} \widetilde{b} = -a \widetilde{x}^2 \end{split}$$

where the last equality follows if we choose

$$\hat{a} = -\hat{a} = \frac{1}{\gamma_a} \tilde{x} \hat{x}$$
 $\dot{b} = -\dot{b} = -\frac{1}{\gamma_b} \tilde{x} u$

Invariant set: $\tilde{x} = 0$.

This proves that $\tilde{x} \to 0$.

(The parameters \tilde{a} and \tilde{b} do not necessarily converge: $u \equiv 0$.)

Suppose the partial system

$$\dot{x}_1 = f(x_1) + g(x_1)\bar{v}$$

can be stabilized by $\bar{v}=\phi(x_1)$ and there exists Lyapunov fcn $V_1=V_1(x_1)$ such that

$$\dot{V}_1(x_1) = \frac{dV_1}{dx_1} \left(f(x_1) + g(x_1)\phi(x_1) \right) \le -W(x_1)$$

for some positive definite function W.

Equation (??) can be rewritten as

$$\dot{x}_1 = f(x_1) + g(x_1)\phi(x_1) + g(x_1)[x_2 - \phi(x_1)]$$

$$\dot{x}_2 = u$$



Introduce new state $\zeta = x_2 - \phi(x_1)$ and control $v = u - \dot{\phi}$:

$$\dot{x}_1 = f(x_1) + g(x_1)\phi(x_1) + g(x_1)\zeta$$
$$\dot{\zeta} = v$$

where

Back-Stepping Lemma

Lemma: Let
$$z = (x_1, ..., x_{k-1})^T$$
 and

$$\dot{z} = f(z) + g(z)x_k$$
$$\dot{x}_k = u$$

Assume $\phi(0) = 0$, f(0) = 0,

$$\dot{z} = f(z) + g(z)\phi(z)$$

stable, and V(z) a Lyapunov fcn (with $\dot{V} \leq -W$). Then,

$$u = \frac{d\phi}{dz} \left(f(z) + g(z)x_k \right) - \frac{dV}{dz}g(z) - (x_k - \phi(z))$$

stabilizes x = 0 with $V(z) + (x_k - \phi(z))^2/2$ being a Lyapunov fcn.

Back-Stepping

Back-Stepping Lemma can be applied recursively to a system

$$\dot{x} = f(x) + g(x)u$$

on strict feedback form.

Back-stepping generates stabilizing feedbacks $\phi_k(x_1, \ldots, x_k)$ (equal to u in Back-Stepping Lemma) and Lyapunov functions

$$V_k(x_1,...,x_k) = V_{k-1}(x_1,...,x_{k-1}) + [x_k - \phi_{k-1}]^2/2$$

by "stepping back" from \boldsymbol{x}_1 to \boldsymbol{u}

Back-stepping results in the final state feedback

$$u = \phi_n(x_1,\ldots,x_n)$$

Step 2 Applying Back-Stepping Lemma on

$$\dot{x}_1 = x_1^2 + x_2$$
$$\dot{x}_2 = x_3$$
$$\dot{x}_3 = u$$

gives

$$u = u_2 = \frac{d\phi_2}{dz} \left(f(z) + g(z)x_n \right) - \frac{dV_2}{dz}g(z) - (x_n - \phi_2(z))$$
$$= \frac{\partial\phi_2}{\partial x_1}(x_1^2 + x_2) + \frac{\partial\phi_2}{\partial x_2}x_3 - \frac{\partial V_2}{\partial x_2} - (x_3 - \phi_2(x_1, x_2))$$

which globally stabilizes the system.

Consider $V_2(x_1, x_2) = V_1(x_1) + \zeta^2/2$. Then,

$$\begin{split} \dot{V}_2(x_1, x_2) &= \frac{dV_1}{dx_1} \bigg(f(x_1) + g(x_1)\phi(x_1) \bigg) + \frac{dV_1}{dx_1} g(x_1)\zeta + \zeta v \\ &\leq -W(x_1) + \frac{dV_1}{dx_1} g(x_1)\zeta + \zeta v \end{split}$$

Choosing

$$v = -\frac{dV_1}{dx_1}g(x_1) - k\zeta, \qquad k > 0$$

gives

$$\dot{V}_2(x_1,x_2) \leq -W(x_1) - k\zeta^2$$

Hence, x = 0 is asymptotically stable for (??) with control law $u(x) = \dot{\phi}(x) + v(x)$.

If V_1 radially unbounded, then global stability.

Strict Feedback Systems

Back-stepping Lemma can be applied to stabilize systems on strict feedback form:

$$\begin{split} \dot{x}_1 &= f_1(x_1) + g_1(x_1) x_2 \\ \dot{x}_2 &= f_2(x_1, x_2) + g_2(x_1, x_2) x_3 \\ \dot{x}_3 &= f_3(x_1, x_2, x_3) + g_3(x_1, x_2, x_3) x_4 \\ &\vdots \\ \dot{x}_n &= f_n(x_1, \dots, x_n) + g_n(x_1, \dots, x_n) u \end{split}$$

where $g_k \neq 0$

Note: x_1, \ldots, x_k do not depend on x_{k+2}, \ldots, x_n .

Example

Design back-stepping controller for

$$\dot{x}_1 = x_1^2 + x_2, \quad \dot{x}_2 = x_3, \quad \dot{x}_3 = u$$

Step 0 Verify strict feedback form Step 1 Consider first subsystem

$$\dot{x}_1 = x_1^2 + \phi_1(x_1), \quad \dot{x}_2 = u_1$$

where $\phi_1(x_1)=-x_1^2-x_1$ stabilizes the first equation. With $V_1(x_1)=x_1^2/2,$ Back-Stepping Lemma gives

$$u_1 = (-2x_1 - 1)(x_1^2 + x_2) - x_1 - (x_2 + x_1^2 + x_1) = \phi_2(x_1, x_2)$$

$$V_2 = x_1^2/2 + (x_2 + x_1^2 + x_1)^2/2$$

Control problems where there is a mixture between continuous states and discrete state variables.

Continuous states: position, velocity, temperature, pressure

Discrete states: on/off variables, controller modes, loss of actuators, loss of sensors, relays, etc

Discontinuous differential equations

Much active field, much left to understand

Control law that switches between different modes, e.g. between

- Time optimal control during large set point changes
- ► Linear control close to set point

Aircraft Example





Phase Plane

No common quadratic Lyapunov function exists.

$$A_1 = \begin{bmatrix} -5 & -4 \\ -1 & -2 \end{bmatrix} \qquad \qquad A_2 = \begin{bmatrix} -2 & -4 \\ 20 & -2 \end{bmatrix}$$

(Branicky, 1993)

Piecewise quadratic Lyapunov functions

$$(x) = \begin{cases} x^* P x & \text{if } x_1 < 0\\ x^* P x + \eta x_1^2 & \text{if } x_1 \ge 0 \end{cases}$$

The matrix inequalities

V

 $\begin{array}{rcl} A_1^*P + PA_1 &< & 0 \\ P &> & 0 \\ A_2^*(P + \eta E^*E) + (P + \eta E^*E)A_2 &< & 0 \\ P + \eta E^*E &> & 0 \end{array}$

with $E = [1 \ 0]$, have the solution $P = \text{diag}\{1,3\}, \eta = 7$.

Flower Example



