

Approximation of Analog Controllers, PID Control

Real-Time Systems, Lecture 8

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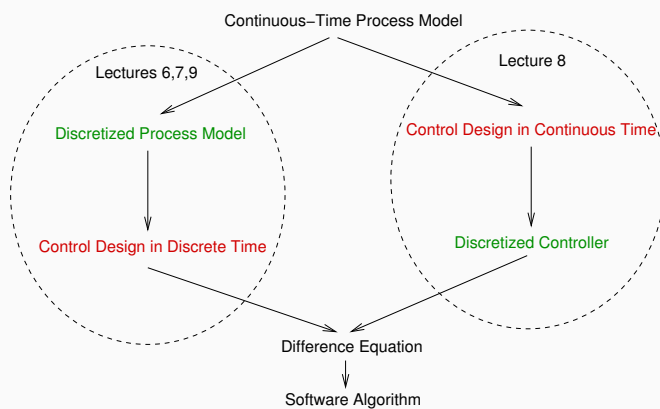
Lecture 8

[IFAC PB Chapters 6–7, RTCS Chapter 10]

- Discrete-time approximation of continuous-time controllers
 - Differential equations
 - State-space systems
 - Transfer functions
- The PID controller

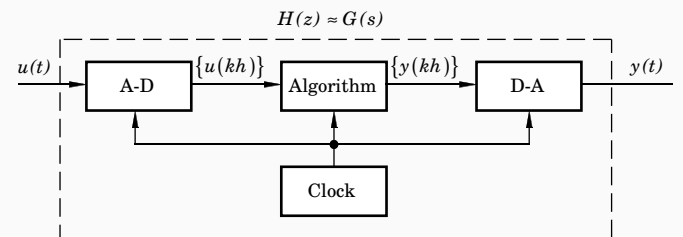
1

Design Approaches for Computer Control



2

Digital Implementation of an Analog Controller



Controller $G(s)$ is designed based on analog techniques

We want to find digital algorithm such that

$$\text{A-D} + \text{Algorithm} + \text{D-A} \approx G(s)$$

NOTE: Involves approximation!

3

Approximation Methods

- Difference and Tustin approximations
- Step invariance approximation (ZOH)
- Ramp invariance approximation (FOH)
- (Pole-zero matching)

(Tustin and the three last methods are available in Matlab's c2d command)

4

Difference and Tustin Approximations of Differential Equations

Forward difference (explicit Euler, Euler's method):

$$\frac{dx(t)}{dt} \approx \frac{x(k+1) - x(k)}{h} = \frac{q-1}{h} x(k)$$

Backward difference (implicit Euler):

$$\frac{dx(t)}{dt} \approx \frac{x(k) - x(k-1)}{h} = \frac{q-1}{qh} x(k)$$

Tustin's approximation (trapezoidal method, bilinear transformation):

$$\frac{\frac{dx(t+h)}{dt} + \frac{dx(t)}{dt}}{2} \approx \frac{x(k+1) - x(k)}{h}$$

$$\frac{dx(t)}{dt} \approx \frac{2}{h} \cdot \frac{q-1}{q+1} x(k)$$

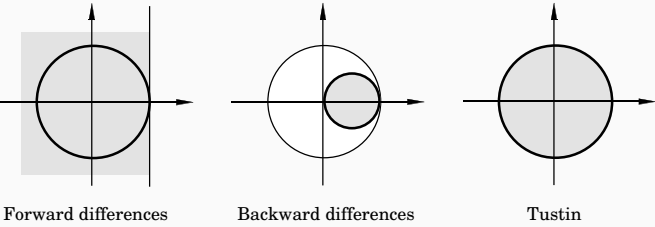
5

Difference Approximations of State-Space Systems	Difference Approximations of State-Space Systems
<p>Assume that the controller is given in state-space form as</p> $\frac{dx}{dt} = Ax + Bu$ $y = Cx + Du$ <p>where x is the controller state, y is the controller output, and u is the controller input.</p> <p>Forward and backward differences suitable for hand calculations</p>	<p>Forward difference:</p> $\frac{dx(t)}{dt} \approx \frac{x(k+1) - x(k)}{h}$ <p>leads to</p> $\frac{x(k+1) - x(k)}{h} = Ax(k) + Bu(k)$ $y(k) = Cx(k) + Du(k)$ <p>which gives</p> $x(k+1) = (I + hA)x(k) + hBu(k)$ $y(k) = Cx(k) + Du(k)$
6	7
Difference Approximations of State-Space Systems	Difference and Tustin Approximations of Transfer Functions
<p>Backward difference:</p> $\frac{dx(t)}{dt} \approx \frac{x(k) - x(k-1)}{h}$ <p>first gives</p> $x(k) = (I - hA)^{-1}x(k-h) + (I - hA)^{-1}hBu(k)$ $y(k) = Cx(k) + Du(k)$ <p>which after a variable shift $x'(k) = x(k-h)$ gives</p> $x'(k+1) = (I - hA)^{-1}x'(k) + (I - hA)^{-1}hBu(k)$ $y(k) = C(I - hA)^{-1}x'(k) + (C(I - hA)^{-1}hB + D)u(k)$	<p>Assume that the controller is given as a transfer function $G(s)$</p> <p>The discrete-time approximation $H(z)$ is given by</p> $H(z) = G(s')$ <p>where</p> <div> $s' = \frac{z-1}{h}$ <p>Forward difference</p> </div> <div> $s' = \frac{z-1}{zh}$ <p>Backward difference</p> </div> <div> $s' = \frac{2}{h} \frac{z-1}{z+1}$ <p>Tustin's approximation</p> </div>
8	9
Example: Discretization	Example: Discretization
<p>Problem: Assume that the following simple controller (filter) has been designed in continuous-time:</p> $U(s) = \frac{1}{s+2}E(s)$ <p>Discretize the controller using forward difference with sampling interval h and write the result as a difference equation.</p> <p>Solution: Replace s with $\frac{z-1}{h}$:</p> $U(z) = \frac{1}{\frac{z-1}{h} + 2}E(z)$ $U(z) = \frac{h}{z-1+2h}E(z)$ $(z-1+2h)U(z) = hE(z)$ $u(k+1) - (1-2h)u(k) = he(k)$ $u(k) = (1-2h)u(k-1) + he(k-1)$	<p>Simulation with $h = 0.1$:</p>
10	11

Properties of the Approximation $H(z) \approx G(s)$

Frequency Distortion

Where do stable poles of $G(s)$ get mapped?



Simple approximations such as Tustin introduce frequency distortion. Important for controllers or filters designed to have certain characteristics at a particular frequency, e.g., a band-pass filter or a notch (band-stop) filter.

Tustin:

$$H(e^{i\omega h}) \approx G\left(\frac{2}{h} \frac{e^{i\omega h} - 1}{e^{i\omega h} + 1}\right)$$

The argument of G can be written as

$$\frac{2}{h} \frac{e^{i\omega h} - 1}{e^{i\omega h} + 1} = \frac{2}{h} \frac{e^{i\omega h/2} - e^{-i\omega h/2}}{e^{i\omega h/2} + e^{-i\omega h/2}} = \frac{2i}{h} \tan\left(\frac{\omega h}{2}\right)$$

Frequency Distortion, Cont'd

Prewarping to Reduce Frequency Distortion

If the continuous-time system affects signals at frequency ω' , the sampled system will instead affect signals at ω where

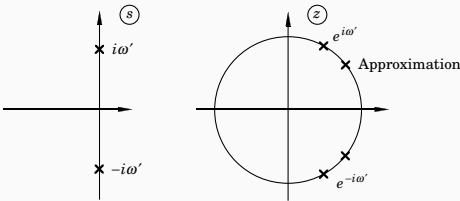
$$\omega' = \frac{2}{h} \tan\left(\frac{\omega h}{2}\right)$$

i.e.,

$$\omega = \frac{2}{h} \tan^{-1}\left(\frac{\omega' h}{2}\right) \approx \omega' \left(1 - \frac{(\omega' h)^2}{12}\right)$$

No distortion at $\omega = 0$

Distortion is small if ωh is small



Choose one point ω_1 . Approximate using

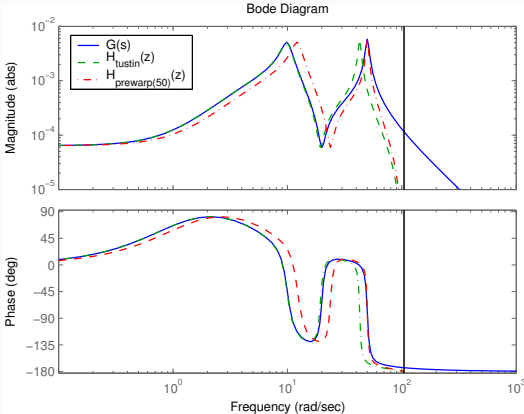
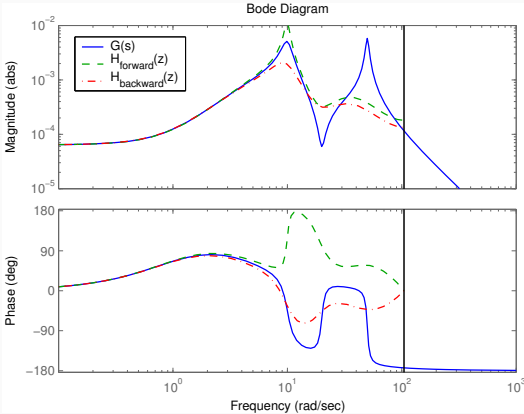
$$s' = \frac{\omega_1}{\tan(\omega_1 h/2)} \cdot \frac{z - 1}{z + 1}$$

This implies that $H(e^{i\omega_1 h}) = G(i\omega_1)$. Plain Tustin is obtained for $\omega_1 = 0$ since $\tan\left(\frac{\omega_1 h}{2}\right) \approx \frac{\omega_1 h}{2}$ for small ω .

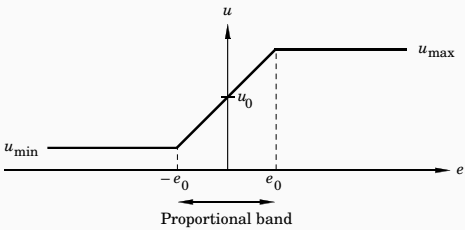
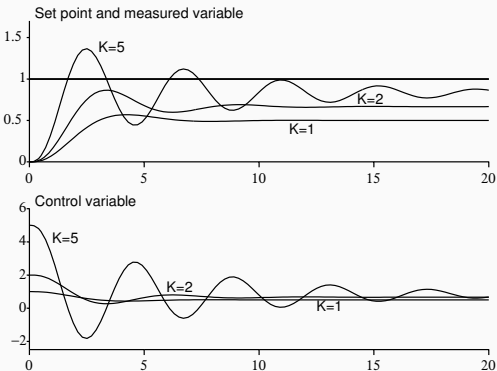
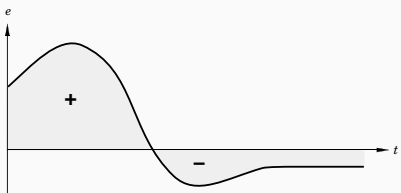
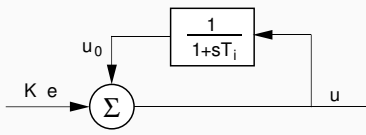
Comparison of Approximations (1)

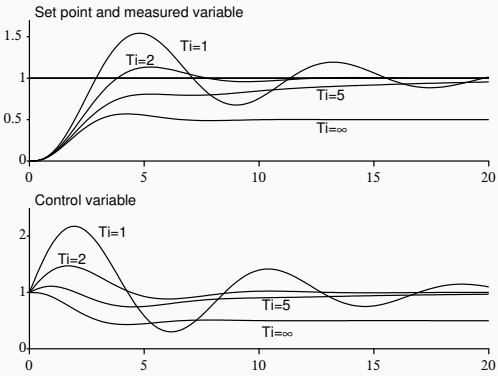
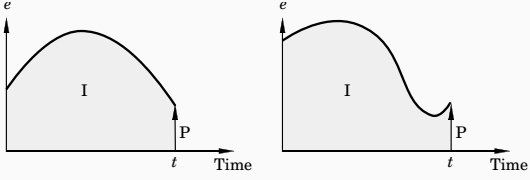
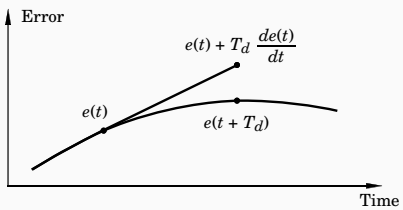
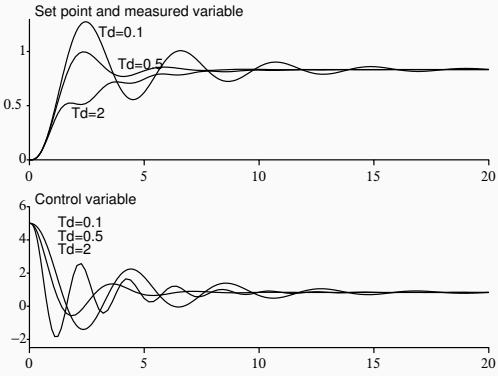
Comparison of Approximations (2)

$$G(s) = \frac{(s + 1)^2(s^2 + 2s + 400)}{(s + 5)^2(s^2 + 2s + 100)(s^2 + 3s + 2500)}, \quad h = 0.03$$



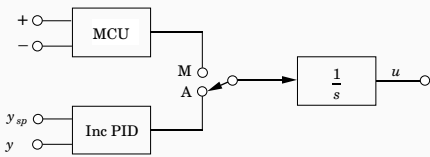
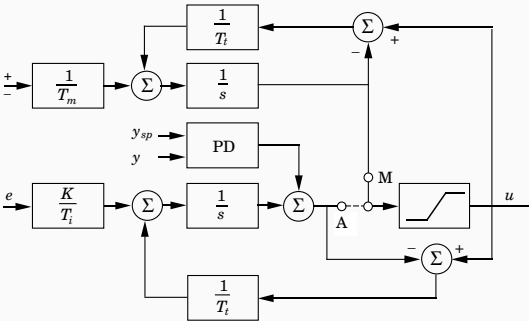
<div data-bbox="36 259 539 291" data-label="Section-Header"> <h2>Step and Ramp Invariance Approximations</h2> </div> <div data-bbox="76 403 694 459" data-label="Text"> <p>Sample the controller in the same way as the physical plant model is sampled</p> </div> <div data-bbox="98 483 657 544" data-label="List-Group"> <ul style="list-style-type: none"> • Step invariance approximation or Zero-order hold sampling • Ramp invariance approximation or First-order hold sampling </div> <div data-bbox="76 568 671 622" data-label="Text"> <p>For a controller, the assumption that the input signal is piece-wise constant (ZOH) or piece-wise linear (FOH) does not hold!</p> </div> <div data-bbox="76 638 713 694" data-label="Text"> <p>However, the ramp invariance approximation method usually gives very good results with little frequency distortion</p> </div> <div data-bbox="766 792 782 810" data-label="Text"> <p>18</p> </div>	<div data-bbox="818 259 1222 291" data-label="Section-Header"> <h2>Comparison of Approximations (3)</h2> </div> <div data-bbox="924 344 1450 757" data-label="Figure"> </div> <div data-bbox="1548 792 1564 810" data-label="Text"> <p>19</p> </div>
<div data-bbox="36 842 121 873" data-label="Section-Header"> <h2>Matlab</h2> </div> <div data-bbox="76 920 639 940" data-label="Text"> <p><code>C2D</code> Converts continuous-time dynamic system to discrete time.</p> </div> <div data-bbox="103 967 759 1198" data-label="Text"> <p><code>SYSD = C2D(SYSC,TS,METHOD)</code> computes a discrete-time model <code>SYSD</code> with sample time <code>TS</code> that approximates the continuous-time model <code>SYSC</code>. The string <code>METHOD</code> selects the discretization method among the following:</p> <ul style="list-style-type: none"> 'zoh' Zero-order hold on the inputs 'foh' Linear interpolation of inputs 'impulse' Impulse-invariant discretization 'tustin' Bilinear (Tustin) approximation. 'matched' Matched pole-zero method (for SISO systems only). <p>The default is 'zoh' when <code>METHOD</code> is omitted. The sample time <code>TS</code> should be specified in the time units of <code>SYSC</code> (see <code>"TimeUnit"</code> property).</p> </div> <div data-bbox="103 1225 767 1337" data-label="Text"> <p><code>C2D(SYSC,TS,OPTIONS)</code> gives access to additional discretization options. Use <code>C2DOPTIONS</code> to create and configure the option set <code>OPTIONS</code>. For example, you can specify a prewarping frequency for the Tustin method by:</p> <pre>opt = c2dOptions('Method','tustin','PrewarpFrequency',.5); sysd = c2d(sysc,.1,opt);</pre> </div> <div data-bbox="766 1375 782 1393" data-label="Text"> <p>20</p> </div>	<div data-bbox="818 842 1211 873" data-label="Section-Header"> <h2>Design Approaches: Which Way?</h2> </div> <div data-bbox="971 913 1402 1178" data-label="Diagram"> </div> <div data-bbox="858 1193 1054 1216" data-label="Text"> <p>Sampled control design:</p> </div> <div data-bbox="880 1236 1362 1375" data-label="List-Group"> <ul style="list-style-type: none"> • When the plant model is already in discrete-time form <ul style="list-style-type: none"> • e.g., obtained from system identification • When the control design assumes a discrete-time model <ul style="list-style-type: none"> • e.g., model-predictive control • When fast sampling not possible </div> <div data-bbox="1548 1375 1564 1393" data-label="Text"> <p>21</p> </div>
<div data-bbox="36 1424 429 1456" data-label="Section-Header"> <h2>Design Approaches: Which Way?</h2> </div> <div data-bbox="189 1496 620 1760" data-label="Diagram"> </div> <div data-bbox="76 1778 343 1800" data-label="Text"> <p>Approximation of analog design:</p> </div> <div data-bbox="98 1818 394 1926" data-label="List-Group"> <ul style="list-style-type: none"> • Empirical control design <ul style="list-style-type: none"> • not model-based • e.g., PID control • Nonlinear continuous-time model </div> <div data-bbox="76 1946 521 1968" data-label="Text"> <p>In most other cases it is mainly a matter of taste.</p> </div> <div data-bbox="766 1957 782 1975" data-label="Text"> <p>22</p> </div>	<div data-bbox="818 1424 962 1456" data-label="Section-Header"> <h2>PID Control</h2> </div> <div data-bbox="858 1579 1492 1659" data-label="Text"> <p><i>Based on a survey of over eleven thousand controllers in the refining, chemicals and pulp and paper industries, 97% of regulatory controllers utilize PID feedback.</i></p> </div> <div data-bbox="858 1680 1088 1702" data-label="Text"> <p>[Desborough Honeywell, 2000]</p> </div> <div data-bbox="880 1753 1133 1848" data-label="List-Group"> <ul style="list-style-type: none"> • The oldest controller type • The most widely used • Much to learn! </div> <div data-bbox="1548 1957 1564 1975" data-label="Text"> <p>23</p> </div>

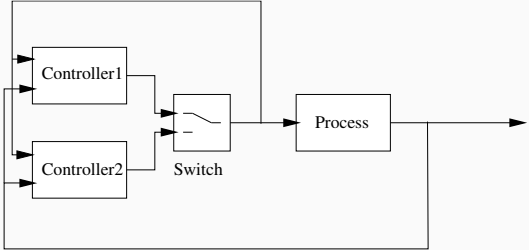
The Textbook Algorithm	Proportional Term
$u(t) = K \left(e(t) + \frac{1}{T_i} \int_0^t e(\tau) d\tau + T_d \frac{de(t)}{dt} \right)$ $U(s) = KE(s) + \frac{K}{sT_i} E(s) + KT_d sE(s)$ $= P + I + D$	 $u = \begin{cases} u_{\max} & e > e_0 \\ Ke + u_0 & -e_0 < e < e_0 \\ u_{\min} & e < -e_0 \end{cases}$
24	25
Properties of P-Control	Stationary Error with P-control
 <ul style="list-style-type: none"> Stationary error Increased K means faster speed, worse stability, increased noise sensitivity 	<p>Control signal:</p> $u = Ke + u_0$ <p>Error:</p> $e = \frac{u - u_0}{K}$ <p>Stationary error removed if:</p> <ol style="list-style-type: none"> $K = \infty$ $u_0 = u$ <p>Solution: Automatic way to obtain u_0</p>
26	27
Integral Term	Automatic Reset
$u = Ke + u_0 \quad (P)$ $u = K \left(e + \frac{1}{T_i} \int e(t) dt \right) \quad (PI)$  <p>Stationary error present $\rightarrow \int e dt$ increases $\rightarrow u$ increases $\rightarrow y$ increases \rightarrow the error is not stationary</p>	 $U = KE + \frac{1}{1 + sT_i} U$ $\left(1 - \frac{1}{1 + sT_i} \right) U = \frac{sT_i}{1 + sT_i} U = KE$ $U = K \left(1 + \frac{1}{sT_i} \right) E$
28	29

Properties of PI-Control	Prediction
 <ul style="list-style-type: none"> Removes stationary error Smaller T_i implies faster steady-state error removal, worse stability 	<p>A PI-controller contains no prediction</p> <p>The same control signal is obtained for both these cases:</p> 
Derivative Part	Properties of PD-Control
 <p>P:</p> $u(t) = K e(t)$ <p>PD:</p> $u(t) = K \left(e(t) + T_d \frac{de(t)}{dt} \right) \approx K e(t + T_d)$ <p>T_d = Prediction horizon</p>	 <ul style="list-style-type: none"> T_d too small, no influence T_d too large, decreased performance <p>In industrial practice the D-term is often turned off.</p>
Alternative Forms	Alternative Forms
<p>So far we have described the direct (position) version of the PID controller in parallel form</p> <p>Other forms:</p> <ul style="list-style-type: none"> Series form $U = K' \left(1 + \frac{1}{sT_i'} \right) \left(1 + sT_d' \right) E$ $= K' \left(1 + \frac{T_d'}{T_i'} + \frac{1}{sT_i'} + sT_d' \right) E$ <p>Different parameter values</p> <p>Less general than the parallel form</p>	<ul style="list-style-type: none"> Incremental (velocity) form $U = \frac{1}{s} \Delta U$ $\Delta U = K \left(s + \frac{1}{T_i} + \frac{s^2 T_d}{1 + sT_d/N} \right) E$ <p>Integration external to the algorithm (e.g. step motor) or internal</p>

Practical Modifications	Limitation of Derivative Gain
<p>Modifications are needed to make the PID controller practically useful</p> <ul style="list-style-type: none"> • Limitations of derivative gain • Derivative weighting • Setpoint weighting • Anti-reset windup 	<p>We do not want to apply derivation to high frequency measurement noise, therefore the following modification is often used:</p> $D(s) = sKT_d \approx \frac{sKT_d}{1 + sT_d/N}$ <p>N = maximum derivative gain, often 5 – 20</p> <p>Another option is to add second-order filter to the measurement signal:</p> $Y_f(s) = \frac{1}{T_f^2 s^2 + 1.4T_f s + 1} Y(s)$ $U(s) = K \left(1 + \frac{1}{sT_i} + sT_d \right) (Y_{sp}(s) - Y_f(s))$ <p>T_f = filter time constant</p>
Derivative Weighting	Setpoint Weighting
<p>The setpoint is often constant for long periods of time</p> <p>Setpoint often changed in steps → D-part becomes very large.</p> <p>Derivative part applied on part of the setpoint or only on the measurement signal.</p> $D(s) = \frac{sT_d}{1 + sT_d/N} (\gamma Y_{sp}(s) - Y(s))$ <p>Often, $\gamma = 0$ in process control (step reference changes), $\gamma = 1$ in servo control (smooth reference trajectories)</p>	<p>Sometimes advantageous to also use weighting on the setpoint.</p> $P = K(y_{sp} - y)$ <p>is replaced by</p> $P = K(\beta y_{sp} - y)$ <p>$\beta \geq 0$</p> <p>Assumes that the I-part is also turned on! (otherwise the controller will try to follow the wrong reference value)</p> <p>A way of introducing feedforward from the reference signal (position a closed loop zero)</p> <p>Improved set-point responses</p>
Setpoint Weighting	Control Signal Limitations
	<p>All actuators saturate. Problems for controllers with integration.</p> <p>When the control signal saturates the integral part will continue to grow – integrator (reset) windup.</p> <p>When the control signal saturates the integral part will integrate up to a very large value. This may cause large overshoots.</p>

<div data-bbox="36 264 263 293" data-label="Section-Header"> <h3>Anti-Reset Windup</h3> </div> <div data-bbox="76 456 279 479" data-label="Text"> <p>Several solutions exist:</p> </div> <div data-bbox="98 501 726 665" data-label="List-Group"> <ul style="list-style-type: none"> • controllers in velocity form (Δu is set to 0 if u saturates) • limited the setpoint variations (saturation never reached) • conditional integration (integration is switched off when the control is far from the steady-state) • tracking (back-calculation) </div> <div data-bbox="766 795 782 813" data-label="Page-Footer"> <p>42</p> </div>	<div data-bbox="818 264 919 293" data-label="Section-Header"> <h3>Tracking</h3> </div> <div data-bbox="858 441 1455 492" data-label="Text"> <p>General technique that can also be used to handle controller mode switches, etc.</p> </div> <div data-bbox="858 510 1313 533" data-label="Text"> <p>The controller output v tracks the actual output u</p> </div> <div data-bbox="880 555 1508 705" data-label="List-Group"> <ul style="list-style-type: none"> • When the control signal saturates, the integral is recomputed so that its new value gives a control signal at the saturation limit • To avoid resetting the integral due to, e.g., measurement noise, the recomputation is done dynamically, i.e., through a LP-filter with a time constant T_t (T_r). </div> <div data-bbox="1548 795 1564 813" data-label="Page-Footer"> <p>43</p> </div>
<div data-bbox="36 846 137 875" data-label="Section-Header"> <h3>Tracking</h3> </div> <div data-bbox="172 898 632 1384" data-label="Diagram"> </div> <div data-bbox="766 1377 782 1395" data-label="Page-Footer"> <p>44</p> </div>	<div data-bbox="818 846 919 875" data-label="Section-Header"> <h3>Tracking</h3> </div> <div data-bbox="858 981 1353 1323" data-label="Figure"> </div> <div data-bbox="1548 1377 1564 1395" data-label="Page-Footer"> <p>45</p> </div>
<div data-bbox="36 1429 197 1458" data-label="Section-Header"> <h3>Discretization</h3> </div> <div data-bbox="76 1583 451 1646" data-label="Text"> <p>Discretize the P, I, and D parts separately Maintains the interpretation</p> </div> <div data-bbox="76 1713 145 1736" data-label="Text"> <p>P-part:</p> </div> <div data-bbox="282 1803 528 1830" data-label="Equation-Block"> $P(k) = K(\beta y_{sp}(k) - y(k))$ </div> <div data-bbox="766 1960 782 1977" data-label="Page-Footer"> <p>46</p> </div>	<div data-bbox="818 1429 979 1458" data-label="Section-Header"> <h3>Discretization</h3> </div> <div data-bbox="858 1489 919 1512" data-label="Text"> <p>I-part:</p> </div> <div data-bbox="1098 1529 1278 1662" data-label="Equation-Block"> $I(t) = \frac{K}{T_i} \int_0^t e(\tau) d\tau$ $\frac{dI}{dt} = \frac{K}{T_i} e$ </div> <div data-bbox="880 1713 1064 1736" data-label="List-Group"> <ul style="list-style-type: none"> • Forward difference </div> <div data-bbox="1090 1744 1329 1798" data-label="Equation-Block"> $\frac{I(t_{k+1}) - I(t_k)}{h} = \frac{K}{T_i} e(t_k)$ </div> <div data-bbox="900 1809 1201 1832" data-label="Equation-Block"> $I(k+1) := I(k) + (K \cdot h / T_i) \cdot e(k)$ </div> <div data-bbox="900 1850 1334 1874" data-label="Text"> <p>The I-part can be precalculated in UpdateStates</p> </div> <div data-bbox="880 1883 1080 1906" data-label="List-Group"> <ul style="list-style-type: none"> • Backward difference </div> <div data-bbox="900 1912 1348 1939" data-label="Text"> <p>The I-part cannot be precalculated, $i(k) = f(e(k))$</p> </div> <div data-bbox="1548 1960 1564 1977" data-label="Page-Footer"> <p>47</p> </div>

Discretization	Discretization
<p>(assume $\gamma = 0$):</p> $D = K \frac{sT_d}{1 + sT_d/N} (-Y(s))$ $\frac{T_d}{N} \frac{dD}{dt} + D = -KT_d \frac{dy}{dt}$ <ul style="list-style-type: none"> Forward difference (unstable for small T_d/large h) Backward difference $\frac{T_d}{N} \frac{D(t_k) - D(t_{k-1})}{h} + D(t_k) = -KT_d \frac{y(t_k) - y(t_{k-1})}{h}$ $D(t_k) = \frac{T_d}{T_d + Nh} D(t_{k-1}) - \frac{KT_d N}{T_d + Nh} (y(t_k) - y(t_{k-1}))$ <p>48</p>	<p>Tracking:</p> <pre> v := P + I + D; u := sat(v, umax, umin); I := I + (K*h/Ti)*e + (h/Tr)*(u - v); </pre> <p>49</p>
Bumpless Transfer	Bumpless Mode Changes
<p>Avoid bumps in control signal when</p> <ul style="list-style-type: none"> changing operating mode (manual - auto - manual) changing parameters changing between different controllers <p>Key Issue: Make sure that the controller states have the correct values, i.e., the same values before and after the change</p> <p>50</p>	<p>Incremental Form:</p>  <p>51</p>
Bumpless Mode Changes	Bumpless Parameter Changes
<p>Direct Position form:</p>  <p>52</p>	<p>A change in a parameter when in stationarity should not result in a bump in the control signal.</p> <p>For example:</p> <pre> v := P + I + D; I := I + (K*h/Ti)*e; </pre> <p>VS</p> <pre> v := P + (K/Ti)*I + D; I := I + h*e; </pre> <p>The latter results in a bump in u if K or T_i are changed.</p> <p>53</p>

Bumpless Parameter Changes	Switching Between Controllers
<p>More involved situation when setpoint weighting is used. The quantity $P + I$ should be invariant to parameter changes.</p> $I_{new} = I_{old} + K_{old}(\beta_{old}y_{sp} - y) - K_{new}(\beta_{new}y_{sp} - y)$ <div>54</div>	 <p>Similar to changing between manual and auto</p> <p>Let the controllers run in parallel</p> <p>Let the controller that is not active track the one that is active.</p> <p>Alternatively, execute only the active controller and initialize the new controller to its correct value when switching (saves CPU)</p> <div>55</div>
PID Code	[Java] Class SimplePID
<p>PID-controller with anti-reset windup and manual and auto modes ($\gamma = 0$):</p> <pre> y = yIn.get(); e = yref - y; D = ad * D - bd * (y - yold); v = K*(beta*yref - y) + I + D; if (mode == auto) u = sat(v,umax,umin) else u = sat(uman,umax,umin); uOut.put(u); I = I + (K*h/Ti)*e + (h/Tr)*(u - v); if (increment) uinc = 1; else if (decrement) uinc = -1; else uinc = 0; uman = uman + (h/Tm) * uinc + (h/Tr) * (u - uman); yold = y; </pre> <p><i>ad</i> and <i>bd</i> are precalculated parameters given by the backward difference approximation of the D-term.</p> <div>56</div>	<pre> public class SimplePID { private double u,e,v,y; private double K,Ti,Td,Beta,Tr,N,h; private double ad,bd; private double D,I,yOld; public SimplePID(double nK, double nTi, double NTd, double nBeta, double nTr, double nN, double nh) { updateParameters(nK,nTi,nTd,nBeta,nTr,nN,nh); } public void updateParameters(double nK, double nTi, double NTd, double nBeta, double nTr, double nN, double nh) { K = nK; Ti = nTi; Td = nTd; Beta = nBeta; Tr = nTr; N = nN; h = nh; ad = Td / (Td + N*h); bd = K*ad*N; } } </pre> <div>57</div>
[Java] Class SimplePID	[Java] Extract from Regul
<pre> public double calculateOutput(double yref, double newY) { y = newY; e = yref - y; D = ad*D - bd*(y - yOld); v = K*(Beta*yref - y) + I + D; return v; } public void updateState(double u) { I = I + (K*h/Ti)*e + (h/Tr)*(u - v); yOld = y; } } </pre> <div>58</div>	<pre> public class Regul extends Thread { private SimplePID pid; public Regul() { pid = new SimplePID(1,10,0,1,10,5,0.1); } public void run() { // Other stuff while (true) { y = getY(); yref = getYref(); u = pid.calculateOutput(yref,y); u = limit(u); setU(u); pid.updateState(u); // Timing Code } } } </pre> <div>59</div>